MATHEMATICAL SIMULATION OF

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MACHINE TOOL STRUCTURES

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1 - SUMMARY:

Machine tool structure is the principal part of the machine which contributes to its dynamic behaviour. For this reason, the study of the different alternative design configurations of the structure will be necessary, and the mathematical simulation of such structure becomes an effective tool.

In this work, the dynamic characteristics of the structure of a horizontal knee-type milling machine are investigated. Basing upon the classical beam theory, a mathematical model for the structure is introduced. With the aid of this model, fourteen suggested design solution are studied and compared with each other. The suggested fourteen structures are of the same height, main dimensions, wall thickness and mass.

The study results in predicting the most suitable structure. Thereby a considerable improvement of the dynamic behaviour of the considered machine is achieved.

2 - INTRODUCTION:

Due to the development of digital computers, several analytical techniques have been adapted for predicting the static and dynamic behaviour of machine tool structures. In fact, these techniques are based on simulating the structure of the machine by a mathematical mode. Thereby

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the different influencing parameters could be inter-related and their effects could be investigated. Applications of such techniques to some types of machine tools are found in the literature (1-9).

In this work, one of these techniques, namely the classical beam theory, is used to investigate the effects of the structural configuration on its dynamic behaviour. The structure of the horizontal knee-type milling machine is chosen, and the different possible variations in its shape are studied.

3 - THE MATHEMATICAL MODEL:

Fig. (1) shows the general shape structural main parts and the main dimensions of the considered horizontal kneetype milling machine. The basic model of the machine is shown in Fig. (2). To adapt such structure to beam equations the structure should be replaced by an idealized model Fig. (3), which is in turn transformed to the mathematical model shown in Fig. (4).

4 - DERIVATION OF THE FREQUENCY EQUATION:

As shown in Fig. (4), the mathematical model of the machine comprises five beams. Accordingly the general solution of the beam equation (10) could be written in the following general form:

$y_r = C_{1r} \sin \frac{2_r}{L_r} x_r$	+ C _{2r}	$\cos \frac{z_r}{L_r} \kappa_r$	+C _{3r} sinh	$\frac{Z_r}{L_r} x_r + C_{4r}$	cosh _L rxr
					(1)

Substituting the end conditions of the different beams into equation (1) yields:-

c ₂₁	+ C ₄₁	= 0	
с ₁₁	+ c ₃₁	≠ Ç	(3)
sin	24C14+cos	Z4C24-sinh	Z_4C_{34} -cosh $Z_4C_{44} = 0$ (4)
cos	Z ₄ C ₁₄ -sin	Z_Ccosh	$Z_{4}C_{34} = \sinh Z_{4}C_{44} = 0$ (5)

 $z_{10} z_{10} z_{10}$ " "²M_c(ain 2²C₁₂+cos 2²C₂₂ + sinh 2²C₃₂+cosh 2²C₄₂)(15) E 1342 (-cos 25C12 + sin 25C22 + cosp 25C32 + cosp 25C42) $- D^{52} C^{52} + D^{52} C^{42} = 0$ (14) $z \uparrow z_2^{C_{12}} + \cos z_2^{C_{22}} - sin z_2^{C_{12}} - \cos z_2^{C_{12}}$ is 20 + $\cos x^{5} + c^{45}$) = $B I^{2} + c^{52} + c^{42}$ E 13 42 (-270 Z C15 - COE Z C22 + E STUP Z C35 -B25C15 - B25C35 * 0 (ET) ******* ces 22²12 + cosh 22²32 + cosh 22²32 + cosh 22²32 + cosh 22²32 which could be written in the torm: (SE2 + ST2) S# # (cos Z₂C₁₂ - sin Z₂C₂₂ + cosh Z₂C₃₂ + sin Z₂C₄₂) c⁵² + c⁴² = 0 (72) ****** $(*C^{2})^{+} = E I^{+} a^{2} (-C_{1} + C_{3})^{+}$ (ττ) · · · · · · E $1^{3}g_{1}^{2}(-\cos z^{3}c^{13} + \sin z^{3}c^{53} + \cos z^{3}c^{33}$ + cosy $S^{3}C^{43}$) = E I⁴ S^{4}_{c} (- C^{54} + C^{44}) (οτ) · · · · · · · E 132 (121 230 - 202 230 - 210 230 - 210 233 + 210 233 (6) *** (*E)+*I) *e=(E*)EZ 4418+EE)EZ 4500+EZ)EZ 418-EI)EZ 50) E (9) $0 = 2^{3}C_{25}$ usos $2^{5}C_{25}$ uis

 $a_{2}(C_{12} + C_{32}) = a_{3} (C_{13} + C_{33})$ which could be written in the following form: $C_{12} + C_{32} - B_{23} C_{13} - B_{23} C_{33} = 0 \qquad \dots \dots \dots \dots \dots \dots (19)$ $E I_{1} a_{1}^{2} (-\sin Z_{1}C_{11} - \cos Z_{1} C_{21} + \sinh Z_{1} C_{31} + \cosh Z_{1}C_{41})$ $-E I_{2} a_{2}^{2} (-C_{22} + C_{42}) = E I_{3} a_{3}^{2} (-C_{23} + C_{43}) \dots \dots (20)$ $E I_{2} a_{1}^{3} (-\cos Z_{1}C_{11} + \sin Z_{1} C_{21} + \cosh Z_{1} C_{31} + \sinh Z_{1}C_{41})$ $-E I_{2} a_{2}^{3} (-C_{12} + C_{32}) = -M_{1} w^{2} (C_{22} + C_{42}) \dots \dots (21)$

Equations (8), (10), (9), (11), (16), (20), (18) and (21) could be replaced by equations $(\overline{8})$, $(\overline{10})$ by combining each two successive equations. This gives:-

 $\sin z_{3}C_{13} + \cos z_{3}C_{23} + \beta_{34}C_{24} + \frac{\alpha}{34}C_{34} = 0 \quad (\overline{8})$

$$\sinh z_{3}c_{33} + \cosh z_{3}c_{43} + \frac{\alpha}{34}c_{24} + \beta_{34}c_{44} = 0 \quad (\overline{10})$$

$$\cos z_{3} C_{13} - \sin z_{3} C_{23} + \Delta_{34} C_{14} + \delta_{34} C_{34} = 0$$
(9)

$$\cosh z_3 C_{33} + \sinh z_3 C_{43} + \delta_{34} C_{14} + \Delta_{34} C_{34} = 0$$
 (11)

sin $z_1 c_{11} + \cos z_1 c_{21} + \beta_{12} c_{22} + \alpha_{12} c_{42}$

$$- D_{12} D_{23} C_{23} = 0$$
 (16)

$$\sinh z_1^{C_{31}} + \cosh z_1^{C_{41}} + \frac{\alpha_{12}^{C_{22}} + \beta_{12}^{C_{42}}}{+ p_{12}^{P_{23}^{C_{23}}}} = 0 \qquad (\overline{20})$$

$$\cos z_{1}c_{11} - \sin z_{1}c_{21} + \frac{A_{12}c_{12} - R_{1}z_{1}c_{22}}{+ \aleph_{12}c_{32} - R_{1}z_{1}c_{42}} = 0$$
(18)

 $\cosh z_{1}C_{31} + \sinh z_{1}C_{41} - \aleph_{12}C_{12} + R_{1}z_{1}C_{22} + \frac{\Lambda_{12}C_{32} + R_{1}z_{1}C_{42}}{12C_{32} + R_{1}z_{1}C_{42}} = 0 \quad (\overline{21})$

where;

$$a_{r} = \frac{L_{r}}{L_{r}}$$

$$B_{r(r+1)} = \frac{a_{r+1}}{a_{r}}$$

$$D_{r(r+1)} = B_{r(r+1)}^{2} \cdot \frac{I_{r+1}}{I_{r}}$$

$$\mathbf{a}_{r(r+1)} = 0.5 \ (D_{r(r+1)} - 1)$$

$$\mathbf{a}_{r(r+1)} = -(\mathbf{a}_{r(r+1)} + 1) = -0.5 \ (D_{r(r+1)} + 1)$$

$$\mathbf{a}_{r(r+1)} = \mathbf{a}_{r(r+1)} \cdot B_{r(r+1)}$$

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$$\mathbf{a}_{r(r+1)} = \mathbf{a}_{r(r+1)} \cdot B_{r(r+1)}$$

$$\mathbf{a}_{r(r+1)} = \frac{\mathbf{a}_{r(r+1)}}{2m_{1}}$$

Simplifying equation (15) by the following substitution:

$$U = \cos Z_{2} + R_{2} Z_{2} \sin Z_{2}$$

$$V = -\sin Z_{2} + R_{2} Z_{2} \cos Z_{2}$$

$$W = -\cosh Z_{2} + R_{2} Z_{2} \sinh Z_{2}$$

$$X = -\sinh Z_{2} + R_{2} Z_{2} \cosh Z_{2}$$

where

$$\frac{m_5 w^2}{E I_2 a_2^3} = \frac{m_5}{m_2} 2_2 = R_2 2_2$$

gives

$$U C_{12} + V C_{22} + W C_{32} + X C_{42} = 0$$
(15)

Equations from (2) to (7), from (8) to (11), from (12) to (14), (15), (16), (17), (18), (19), (20) and (21) form a set of equations containing the unknown constants C_{11} , C_{12} , ..., C_{45} and their unknown coefficients. Since the values of these constants are not zero, hence the set of equations yield to the frequency equation in the form of 20th order deterimant, after reduction to 17th order and rearrangement the determinant equation (22) is established.

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The values of Z_2 , Z_3 , Z_4 and Z_5 could be determined in terms of Z_1 from the equation (2):

$$z_{r+1} = z_r \cdot \frac{L_{r+1}}{L_r} = B_{r(r+1)}$$

Substituting these values into equation (22) yields the frequency equation in terms of Z, alone.

5 - THE NATURAL FREQUENCIES AND MODERFORMS OF THE CONSIDERED MACHINE:

The different natural frequencies of the machine could be determined from the following equation⁽²⁾:

$$f = \frac{z_1^2}{2 \chi L_1^2} / \frac{E I_1 g}{P A_1} CPS$$

by substituting the different values of (Z_1) . The corresponding modeforms of the machine could be determined by calculating the deflection of each beam using equation (1). For this purpose the values of C_{11} , C_{21} , ..., C_{45} should be evaluated. This could be done by solving the set of equations given in section (4). This set of equations could be first reduced to nineteen equations by omitting the last equation and substituting $C_{11} = 1$. The moderform is only found on a relative basis, so that all the modeform is only found on a relative basis, so that all the modeform equation (1) can be multiplied on the right-hand side by the same arbitrary constant $\neq 0$.

6 - THE BASIC MODEL OF THE MACHINE AND THE SUGGESTED MODIFICATIONS:

The basic model of the machine is shown in Fig. (2). Four modifications are suggested and shown in Figs (5), (6), (7) and (8). All modifications are done in such a way that the column of the machine has the same mass, height and wall thickness, whilst the dimensions and weights of the table and overarm are kept constant. In the first modification the column has vertical sides and tapered back, whilst in the second the column assumes squared cross-section. In the third and fourth modifications the back side wall is given a circular shape. The idealized models of the four suggested modifications are shown in Figs. (9), (10), (11) and (12).

The configuration of the overarm of the machine is also investigated. Five different cross-sectional alternative shapes of the overarm are suggested and shown in Fig. (13). The five suggested overarms have the same mass and length.

The investigation of the dynamic behaviour of the basic and suggested models will be done in the following way:-

- 1. The natural frequencies and modeforms of the basic model will be first determined and considered as basic data.
- 2. The natural frequencies and moderforms of each of the suggested modifications will be measured and compared with the basic data in order to evaluate their effects. In this way the suggested model which will have the superior properties will be detected.
- 3. The dynamic behaviour of the superior model will be further investigated by changing its overarm in the sequence shown in Fig. (13).

7 - RESULTS, DISCUSSION AND CONCLUSIONS:

The results of this investigation are tabluated and plotted in the form shown hereafter.

Table (1) shows the characteristics of the basic model of the machine, while tables (2) to (5) show the characteristics of each suggested modification. Tables (6) to (9) give the results of comparison between the characteristics of the suggested modifications and the basic model, which show that the third and fourth suggested modified models are superior than the basic.

The characteristics of the third modified model when equipped with the different suggested shapes of overarms, shown in Fig. (13), are included in tables (10) to (14). Tables (15) to (19) show the results of comparison between the characteristics of this model, when equipped with the different overarm shapes, with its basic model. The basic model in this case will be the third modified model equipped with the original overarm as it is shown in Fig. (7). Hence the results given in table (4) will be the basic data. Tables (15) to (19) show that the third modification equipped with overarm possessing the second and/or the fifth shapes gives the best results. These two combinations will be termed (III - 2) and (III - 5).

Tables (20) to (24) are concerned with the characteristics of the fourth modification when equipped with the different suggested shapes of overarms, whilst tables (25) to (29) give the results of comparison between its characteristics and that of its basic model. The basic model in this case will be the fourth modification equipped with the original overarm as it is shown in Fig. (8), and its data given in table (5) will be considered as basic data. The results of comparison show that the fourth modification equipped with overarm possessing the second and/or the fifth shapes gives the best results. These two combinations will be termed (IV - 2) and (IV - 5).

The main results of this investigation are summarized in Figs (14) to (18). These figures compare the characteristics of the basic model of the machine with those of the four best models, namely, (III - 2), (III - 5), (IV - 2) and (IV - 5). Regarding the moderforms, the best four models behave approximately in the same manner, as it is shown in Fig. (14), and from Figs (15) to (17) which show the deflections of the main parts of each model at the different modes. However, the modified model (III - 2) may be the best one, in the sense that, it ensures, the minimum deflection and hence the maximum rigidity. The percentage reduction in the deflection of the main parts of that model is given in Fig. (18), which shows how far the deflection of each part of the suggested model is reduced in comparison with those of the basic model.

The percentage reduction reachs 21% for the column, 23% for the table and 71% for the overarm. Further investigation on prototype scale seems to be necessary in order to indentify securely the suggested model which could be recommended.

Regarding the magnitude of the natural frequencies, it could be seen that, the first natural frequencies of the best model

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range from 203 to 221 C/S compared to 181 C/S for the basic model of the machine. This gives a mean increase of about 31 C/S corresponding to 1860 r.p.m., which makes the first natural frequencies of the suggested models far away, with an example space, from the working range of the machine.

In order to visualize the individual effects of the column and hence the overarm, the set of results illustrated in Figs (19) to (22) is presented. Figs (19) to (20) compare the deflections of the different main parts of the basic, third and fourth suggested models when equipped with the original overarm. On the other hand, Fig. (22) indicates the percentage reduction reachs 4% for the column, 9% for the table and 28% for the overarm.

From the presented results and discussion the following main conclusions could be drawn:-

- When the model mass and main dimensions are kept constant, the rigidity of a machine tool structure could be increased by altering its configuration. The optimum configuration could be detected by checking the various alternative possible shapes of its main parts.
- 2. For the purpose of checking the different possible configurations of a machine tool structure, the classical beam theory may be one of the useful tools.
- 3. The four suggested models for the considered milling machine exhibit reduced deformation and increased natural frequency.
- 4. The percentage reduction in deformation of the suggested models amounts to 21% for the column, 23% for the table and 71% for the overarm. The natural frequency increased by about 31 C/S.
- 5. Rounding the back-side of the column and/or the overarm yield considerable improvement of the dynamic characteristics of the milling machine.
- 6. Rounding the back-side of the column of the horizontal milling machine while keeping the other parts having the original shapes results in reducing the deflection by about 4% for the column, 9% for the table and 28% for the overarm.

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NOMENCLATURE

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A _r	Cross-sectional area.
a r	Frequency constant.
$B_{r(r+1)}$	Area constant.
c _{1r} ,c _{2r} ,,c	Boundary constants.
E	Modulus of elasticity.
E I .	Flexural rigidity.
f	Frequency.
g	Acceleration of gravity.
I _r í	Moment of inertia of area.
Lr	Length.
mr	Mass.
R_1, R_2	Masses ratio.
r	Subscript taking the values 1,2,,5 and
	indicates the number of beam.
w	Circular frequency.
× _r	Abscissa.
y _r	Ordinate.
	frequency constant.
-	Density of beam material.
P A	Mass per unit length.

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	Beam 5 X /I	0 0.25 0.5 0.75 1	0 4.51 14.22 26.67 39.98	9 0 0.18 0.09 -0.17 -0.51	9 0 -1.02 -2.08 -1.56 0.38	model.	-63-					Third mode f = 860 C/S
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	Beam 4	X4/L4	0 0.25 0.5 0.75 1	1.15 1.44 1.74 2.03 2.33	0.79 1 1.22 1.43 1.65	-1.07 -1.52 -2.02 - 2.55 - 3.1	of the fourth mov		1		·	
Deflections	Béam 3	X3/L3	0 (0.25 0.5 0.75 1	0 6.28 0.57 0.66 1.15	0 0.19 0.38 0.59 0.79	0 -0.13 -0.37 -0.69 -1.07	ıral fraquencies	Initial position. Modeforms.				Second mode
	Beam 2	X2/L2	0 0.25 0.5 0.75 1	1.25 1.87 2.59 3.4 4.3	5 0.95 1.33 1.69 2.03 2.35	9 0.59 0.46 0.35 0.25 0.15	Jeforms and nat					
	Beam 1	X1/L	0 0.25 0.5 0.75 1	5 0 0.08 0.33 0.73 1.25	.7 0 0.08 0.29 0.59 0.99	3 0 0.12 0.38 0.59 0.5	Table(5) Moc		Γ			Fundamental mode
-0	⊶ ۱ ۱	0 ² c/s	W	St 0.62 163.	2 1.06 306.	3 1.8 864.						

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		Beam 5	X ₅ /L ₅	0.25 0.5 0.75 1	0.75 0.75 0.75	0.33 2.79 4.47 2.63	-0.33 -0.37 -0.3 -0.73	-68 -		Beam 5	X5 /Ls
.*				0	0	36 0	0	del .			
	A second way before a management of the second s	Beam 4	X 4 /L 4	0.25 0.5 0.75 1	4 0.94 0.95 0.95 0.9	8 0.88 0.88 0.87 0.8	2 1.18 1.15 1.13 1.1	und basic mo		Beam 4	X 4 /L 4
9. s. j.	os			0	6.0 26.0	0.88 0.8	1.22 1.2	tion c	S	~	
	flection rati	Beam 3	X ₃ /L ₃	0 0.25 0.5 0.75	0 0.93 0.93 0.93	0 0.9 0.9 0.89	0 1.7 1.29 1.22	lırst modifica	flection ratic	Beam	X 3 /L 3
	Del	Beam 2	X ₂ /L ₂	0.25 0.5 0.75 1 0	22 1.01 1.01 1.01 1.01	01 1 0.98 0.97 0.95	96 0.68 0.46 0.21 -0.15	son between the f	;ət	Beam 2	X,/L,
		Beam 1	X, /L,	0.25 0.5 0.75 1 0	1 1.03 1.03 1.02 1.0	1 1:03 1:01 1:01 1:0	1.27 1.2 1.12 0.96 0.	ole(6) Comparis		Beam 1	X. /L ,
n de da la composition Regioner de recenter		-soi	LDI	0	0 03	66 0	0 68	Tab		.soi	101
		Mode Mode	nbə I		1 mode 1.	2 mode 0-	3 mode ^{0.1}			Mode 5	50

-1.34 1.23 0.9 0.91 0.44 1.47 -0.54 0 0.25 0.5 0.75 0.93 0.91 -0.57 -0.59 0.69 0 0 0 0.93 0.98 0.98 0.98 ____ 1.2 0.75 1 1.21 0.93 0.93 1.22 0.25 0.5 1.24 1.24 1.23 0.94 0.94 1.06 0.98 0.98 0.98 0.98 0.99 0 -------1.29 1.07 0.75 0.95 1.32 0.5 0.95 1.5 **...** 0.25 0 o ¢ 0 1.02 1.02 1.03 660 0.69 1.58 -0.75 -80 0.5 -0.86 0.25 1.02 1.02 1-0-1 -101 0 1.03 1.02 101 101 ---0.75 1.14 1.09 1 1.03 1.03 0.5 1.18 0.25 ο ο 0 o ⁵model 0.95 2 mode 0.95 3 mode 1.04 З

Table(7) Comparison between the second modification and basic model

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Table (393) Comparison between the fourth modification and basic model.

3 rd mode	2 mode	1 st mode		number	Mode	
1-03	0.97	1.0 <u>1</u>	F	re qu rai	Jenc ios.	У
0	0	0	0			
1.09	-	69.0	0.25		m	
1.08	1-03	0.97	0.5	1	ēq	
1.03		0.95	0.75	Ľ	ת →	
	660	0.97	-			
	66.0	0.97	0			
0.9	0.99	0.97	0.25	×	m	
0.85	0.98	0.96	0.5	2/2	ěq	
0.89	0.97	0.96	0.7 5		n 2	
1.25	0.97	0.96	-			
o	0	0	0)eil
1.3	56-5	6.03	C.25	×	m	ec ti
1.19	0.92	0.93	0.5	$\left[\right]$	ear	Ŋ
1.17	C 9 3	6.93	0.75		ι Ξ	rati
71.1	5.6'0	0.93	-			so
1.14	0.93	0.93	0			
1.13	0.92	0.93	0.25	×	m	
1.12	0.93	0.93	0.5	2	ěq	
1.12	0.92	0.93	0.75		4 L	
1.1	0.92	0.93		ļ		
o	0	0				
0.04	0.27	0.83	0.25	\mid	B	
0		0.82	0.5	5/1-	ăm	
-006	4.76	0.82	0.75	5	თ	
0.39	2.76	0.82	-			

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Table (⁸) Camparison between the third modification and basic model .

	У)efte	ecti	3	ā	los	1						1		{	
Mode	ios.		m	Beai	3				Bec	3	2			Bec	Ĩ				Bec	E.				Bea	3	Ŭ.
number	requ rat		×	2					×,	L ₂			\mathbf{x}	1 5	ய்				\sum	ŕ			\sim	5/1	ۍ ۱	ļ
	F1	<u> </u>	0.25	0 5	0.75	-	٥	0.25	0.5	0.75		0	0.25	0.5	0.75	~	0	0.25	0.5	0.7		0	0.2	0.5	0	5
	.93	0		**		0.99	66.0	66.0	96.0	0.98	0.97	0	0 93	20.9	60	1 60 1	6.0	0.91	60	0.91		0	0.7	7 0.7	4 0.7	0
2 mode	. <u>.</u>	0		1.03		~		0.99	0.97	0.96	0.95	0	4.05	6.0	0.63	280	0.8	9 0.8	0.8	0.8	7 0.8	0	0.1	6-3.7	8 5.4	7 3.0
3 ^{mode}	1-02	•	1.27	1.17	1.09	0.96	96.0	0.72	24.0	1.0				 	12	4 1.19	1.10	1.1	1.1.	1.13	2 1.1	0	ç E	8 0.3	2 0.2	
				Ĩ																						

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-70f = 877 5 C/S 0.15 0.02 -028 -0.67 -0.43-0.63-0.27 0.65 4.16 2.96 24.16 36.11 and a state of the Third mode 0 0.25 0.5 0.75 ഗ Beam X, / Table(10) the third modification equipped with the first overarm 0 - 3.16 - 0.41 - 0.73 - 1.12 - 1.12 - 1.55 - 2.44 - 2.56 - 3.04 0 0.28 0.57 0.87 1..16 1..16 1.45 1.75 2.05 2.34 0 0.08 0.29 0.6 0.97 0.97 0.35 1.72 2.06 7.39 0 C.18 0.37 0.57 0.76 0.76 0.96 1.17 1.37 1.58 0.25 0.5 0.75 1 Beam 4 X4/L 0 c/5 Deflections 5:cond mode C.25 0.5 0.75 Beam 3 and all and a وم /د، Initial position - Modeforms 0.14 0.41 0.61 0.57 0.57 0.37 0.2 0.04 0.13 0 0 0 1.3 1.96 2.73 3.59 4.55 -0.25 0.5 0.75 Beam 2 0 0.09 0.35 0.75 1.3 ---0 0.25 0.5 0.75 Fundamental mode = 161 2 C/S Beam 1 X,/L, 1.89 877.5 0 0 1.07 281.3 0 0.81 161.2 c/s Ň Зd aboM 2nd st 'ou

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	β ard	272	_st	М	od	e r	0.
	1.89	1.09	6.2.9			N	
	877.5	291.9	207.9		c/s	-+ 7	
	0	0	0	0			
	0.14	0.08	80.0	0.25	\times	ω	
1	0.41	0.78	0.31	0-5	1/1	ear	
	0.61	0.58	0.66	0.75		л —	
	0.57	0.92	1.13		÷		
	057	6-97	1.11	0			
	0.37	1.28	1.64	0.25	\mathbf{x}	B	
	0.2	1.61	2.23	2.0	(2/L	Par	
	0.03	1.91	2.68	0.75	Ň	N	
	-01 4	2,17	3.55				
	0	0	0	0	_		
	-016	0.7	0. ?3	0.2 5		B	Def
	0.41).3 5	0.47	0.5	3	ê a n	lec
	-0.74	0.53	0.71	0.75	w	ω	lion
	-1.12	0.71	0.95				S
	-1.12	0.71	0.95	0			
	-1.56	0.9	1.19	0.25	×	B	
	-2.05	1.09	1.44	5.0	1	earr	
	.2.57	1.29	1.69	0.75	*	4	
ł	-305	87-1	1.93				
	0	0	0	0			
	- 6.32	0.05	.56	0.25	×	Be	
	~ 0.5t	-0.55	4.38	0.5		m	
	-0.53	-1-29	7.84	0.75	5	տ	
	-0.13	-2.12	11.51	-			

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	Beam 5 X ₅ /L ₅ 0 0.25 0.5 0.75 1	0 3.51 10.77 19.99 29.82 0 0.03 - 0.35 - 0.95 -1.67 00.4 -0.67 -0.33 0.47	overarm	-77-	Third mode f = 877.5 C/S
	Beam 4 X ₄ /L ₄ 0 0.25 0.5 0.75 1	1.12 1.41 1.7 1.99 2.28 0.75 0.95 1.15 1.35 1.55 -1.12 -1.55 -2.04 -256 -308	with the fourth	i	
Defrections	Beam 3 X ₃ /L ₃ 0 0.25 0.5 0.75 1	0 0.26 0.56 0.84 1.12 0 0.8 0.37 0.56 0.75 0 -0.16 -0.41 -0.73 -1.12	tion equipped	- Mocietor ms.	
	Beam 2 X ₂ /L ₂ 0 0.25 0.5 0.75 1	1.27 1.91 2.64 3.46 4.37 0.96 1.33 1.68 2.01 2.31 0.57 0.37 0.2 0.04 0.13	e third modifica		
	Beam 1 X/L 0 0.25 0.5 0.75 1	2 0 0.09 0.34 0.74 1.27 5 0 0.08 0.29 0.59 0.96 5 0 0.14 0.41 0.61 0.57	Table(13) th		
	z, f	0.83 196. 1.08 286. 1.89 877.			نسب الله

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with its	1911 C
basic model.	
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Table (
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Comparison
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3 rd mode	2 nd mode	1 st mode		number	Mode	
-	1-02	1.23	F	req rd	uen <mark>c</mark> tios.	У
Ö	0	0	0			
-		69.0	0.25	$\mathbf{\mathbf{x}}$	m	
-	0.96	16:0	0,5	1.	Bea	
-	860	68.0	0.75	-	ิ สา	
-	960	0.87	-			
-	0.9 6	0.87	o			
	0.96	0. 86	0.25		œ	
-	0.95	0.84	0.5	1 2	ear	
0.75	0.95	0.82	0.75	2	n 2	
1.07	0.94	0.81				De
0	0	0	0			flec
-	0.94	0.62	0.25		m	tio
-	0.94	0.84	0.5	5	e o	
1.01	0.94	0.84	0.75	Ľ	3	utio
	0.94	0.85	-			S S
	0.94	0.85	0			
	0.95	0.84	0.25		B	ļ
-	0.95	0.85	0.5		Bangar 1	
	0.95	0.85	0.75		4	
-	0.95	0.84	-			
0	0	0	0			
0.82	-1.66	0.45	0.25		Ø	
0.92	0.15	0.41	0.5	5	POT T	
1.6	1.39	0.4	0.75	ե	σ.	
-0.29	1.35	0.34	-			

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the first overarm with its basic model.

				r			3	-76-		_		-	والمناجع والم		
			-	1.13	1.04	1.26						-	0.98	1.02	1.04
	ۍ ا	ب ب	0.75	1.13	1.05	0.68		•		ഹ	ۍ ا	0.75	1.01	õ	-
	ean	2	0.5	1.13	109	1.03				E	2	0.5	1.01	1-06	1.0.1
	р С	×	0.25	1.12	0.65	1.05				ഫ്	×	0.25	1.9	-	1.02
			0	0	0	0	ith					0	o	0	0
		Ī	-	1.02			ג ק						~~	-	***
	4	, 	0.75	1.02	-		bbe			4 7	ۍ ل_	0.75	-	-	
	ean	- 1	0.5	1.02			qui			Sea	1 4	0.5	-	-	
	m	×	0.25	1.02	-	-	e C			ш		0.25		-	-
			0	1.02	-	-	atio					0		-	
S			-	1.02	-	-	ifico	del	S			-	-	-	-
atic	n 3	r.	0.75	1.02	-	-	pou	ů E	ati	3	۲. ا	0.75	*	***	-
r R	ean	(3 /L	0.5	1.02	-	-	p	sic	Б Б	ean	(3/	0.5	~		
	ന		0.25	-	-	•	Ē	βα	ecti	£		C.25		-	- -
e le			0	0	0	0	the	its)efl(,	0	0	0	0
ŏ			1	1.02	-	1	uen 1	/ith		2		-	-		
	n 2	- 2	0.75	1.02	**	-	etwe	s. E		E	- 2	0.75	-	-	
	3ear	(2 A	0.5	1.02	-	-	q	זס		Bea	(2/1	0.5			***
	ш		0.25	1-01		1	isor	оvе				0.25	***	دي 	
			0	5 - Ci	~		par	5 Z				0			
			*	٦. ا		•	l mo	Ţ				-			*-
	F	-	0.75	õ	10.1			the		n 1	<u>_</u>	0.75			
	Seal	<u> </u>	0.5	-	<u>~</u>	-	11			Bear) X	5.0 2			
	ш		0.25	-	-	-	le (•		ш		0.25		جو سيبين	
			0	0	0	0	Tab					0	0	0	0
٨:	anai . 2011	iper Peri	4	0.97	-	-				soit Soit	pan bri	1		بن ه :	
	Mode	number		1 ⁵ hode	2 mode	3 mode			ć	до Моф	number		1 ^s fmode	2 mode	3 mode

the fourth overarm with its basic model.

Table (5^{18}) Comparison between the third modification equipped with

1 st mode 1.2 0 0.49 0.91 0.9 0.49 0.87 0.86 0.83 0 0.86 <t< th=""><th>Frequency ratios.</th><th>0</th><th>0.25 X D</th><th></th><th></th><th></th><th>╺┼╌┼╌┽╌┥╷║</th><th></th><th></th><th>0.5 / M</th><th>22</th><th>-</th><th>• efi</th><th>Bec Bec</th><th></th><th>- 0 W 3 7</th><th>- lio</th><th></th><th></th><th>).25 XG</th><th>0.5</th><th>4</th><th>-</th><th>0</th><th></th><th></th><th>Beam X₅/l</th><th>Beam 5 X₅/L₅</th><th>Beam 5 X₅/L₅ 0.25 0.5 0.75</th></t<>	Frequency ratios.	0	0.25 X D				╺┼╌┼╌┽╌┥╷║			0.5 / M	22	-	• efi	Bec Bec		- 0 W 3 7	- lio).25 XG	0.5	4	-	0			Beam X ₅ /l	Beam 5 X ₅ /L ₅	Beam 5 X ₅ /L ₅ 0.25 0.5 0.75
Ode 1.2 0 0.49 0.91 0.49 0.49 0.49 0.46 0.46 0.43 0 0.46 0.4	F	0	0.25	0,5	0.7	5		0	.25	5	0.75		0	0.25	0.5	0.7).25	0.5	0.75	-		0	0 0.2	0 0.25 0.	0 0.25 0.5 0.	0 0.25 0.5 0.75
$\frac{2^{6}\text{mode}}{3^{7}\text{mode}} 1 0 1 0.96 0.96 0.97 0.97 0.97 0.96 0.95 0.94 0 0.94 0.94 0.94 0.94 0.96 0.96 0.96 0.95 0.95 0.96 0.96 0.95 0.96 0.96 0.96 0.95 0.96 0.96 0.96 0.96 0.96 0.96 0.96 0.96$	ode 1.2	0	680	0.9	<u> </u>	0.0	Q Q		.87	0.86	0.84	0.83	0	0.86	0.80	0	6 0.B	<u> </u>	6	9.86	0.86	0.80	0.86	- <u></u>	Ċ,	0	0.49 0.4	0 0.49 0.45 0	0.49 0.45 0.44
3 rd 1 0 1 1 1 1 1 1 0.75 1.07 0 1 1 1 1 1 1 1 1 1	10de 1.02	0		0.9	6.0	8 0.9	¥7 0	.97	,e.(0.96	0.95	0.94	o	0.94	5.0	6.0	6.0	<u> </u>	-96-	9,9,6	95	360	0.96		0		0 -1.33 1.1	0 -1.33 1.59 1.	0 -1.33 1.59 1.37
	ode 1	o						•		ھم 	0.75	1.07	0						•••	هـ.					0	0 0.8	0 0.84 0.	0 0.84 0.9 2 1	0 0.84 0.92 1.57

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Table (19) Comparison between the third modification equipped with the fifth overarm with its basic model.

							-	-7	8-					
			-	42.14	-0.33	0.1 3							ode S	2/2
		<u>م</u>	0.75	28.11	10.0	60-0				p		 [е , g	r. 4
	E	(s/L	0.5	14.99	0.17	0.02							Thir ee	0
	Bea	×	0.75	72.2	0.2	£070-	ε					·	•	-
			0	٥	0	0	era							
			-	2.43	1.68	1. E 1	ð				h			
		-	0.75	2.12	1.46	-2.56	rst							
	7		0.5	1.82	1.24	-2.02	e 11							
	Bea		0.25	1.51	1.02	-1.52	ţ,							
	ш 		0	1.2	18.0	61-	vith							
			-	1.2	18.0	-1.07	ר ס							•
ons	m	- m	0.75	0.9	0.6	697-	əde	ition					abon C	5
ecti	Ξ	3/	6 6	0 59	0 39	-(37	linb	Sod	гщS	<u> </u>	 <u> </u>	 		5.00x
e ti	Bea	\times	0.25	0.29	0.13	51.0-	े ट	tial	deto			1111	Seco f	4
			0	o	0	0	i i i	1	ž					
			-	4.57	2.43	0.15	ifico							
	5	2	0.75	3.59	2.08	0.25	poc			ľ	۹į			
	ε	2	0.5	2.72	1.72	0.35	ь Ч	i	1					
	Beo		0.25	1.69	.1.35	0.46	but							
			0	1. 29	0.96	0.59	ŭ Ŭ							
			-	1.29	0.96	0.59	ţ							
		-	0.75	0.75	0.59	C.56							ę,	S
	ε		0.5	0.34	0.29	0.38	5						- 2 - C	5
	Bea	\times	0.25	0.09	0.08	0.12	ble			F	 	 	mento	170.3
			0	0	0	0	La		/	1			unda: I	8
	4-	c/s		170.3	300.9	884.3					ŧ.		U.	
		Ň		0.79	1.05	60] /			i				
••	u a	pq	W	st	2nd	3, d				i	• 1			



ഗ X Beam 4 Beam X Deflections ო X₃/ L₃ Beam X2/L2 Beam 2 Beam 2nd 1.07 312.5 0 ท์ 3rd apow ŝ ·ou

	Beam 5	X5/L5	0 4.39 13.82 2.53 28.00	0 0.05 -0.3 -0.85 -1.40	0 -0.04 0 0.08 0.14	verar m	-						Third mode f = 884.3 C/S	
	Beam 4	A la sel a sel a sel a	1.19 1.49 1.79 2.1 2.41	0.76 1.01 1.22 1.44 1.6	-1.07 -1.52 -2.07 -2.55 -3.0	with the third c		-						
Def.ections	Beam 3	X ₃ /L ₃	0 0.29 0.59 0.89 1.19	0 0 19 0.38 0.50 0.76	0	ation squipped	- Initial position	- Modef Irms.		 			5econd mode f = 306.7 C/S	
a de la companya de l	Beam 2	X ₂ /L ₂	1.28 1.93 2.69 3.53 4.48	0.95 1.33 1.69 2.04 2.36	0.59 0.46 0.35 0.25 0.15	e fourth modific								
	Beam 1	X1/L1 0 10 10 10 10 10	0 0.05 0.34 0.74 1.28	0 0.08 0.25 0.59 0.95	0 0.12 0.38 0.59	Table(22) th		/				1111111111111	Fundamental mode { = 174.7 C/S	
		-' C/S	0.8 174.7	1 1.06 306.7	1.8 834.3			-						
0	u aŗ) ON	<u>ñ</u>	ц К	3				ſ					

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	Beam 5	X _s /L _s	0 0.25 0.5 0.75 1	0 1.78 5.14 9.32 13.76	0 0.04 -0.54 -1.3 -2.13	0 -0.03 0.04 0.13 0.2	verarm				 		<u></u>	77777777777777777777777777777777777777	f =884.3 C/S
	Beam 4	X4/L4	0 0.25 0.5 0.75 1	0.99 1.75 1.51 1.77 2.63	2.26 0.96 1.17 1.38 1.59	-1.08 -1.53 -2.04 -2.54 -3.13	with the fifth or			I		ļ			
Deflections	Becim 3	X ₃ /L ₃	0 0.25 0.5 0.75 1	0 0.24 0.49 0.74 0.99	0 0.18 0 37 0.56 0.76	0 -0.13-0.37 -0.7 -1.08	ation equipped	nitial position	fadelor ms .		 •			Consider made	f = 312,5 C/S
	Beam 2	X,/L,	0 0.25 0.5 0.75 1	1.11 1.64 2.24 2.88 3.58	0.92 1.29 1.63 1.94 2.22	0.58 0.45 0.35 0.25 0.15	he fourth modific								
	E E E E	X /1.	0 0.25 0.5 0.75 1	0 0.08 0.3 0.65 1.11	0 0.07 0.28 0.57 0.92	0 0.12 0.38 0.58 0.58	Table (24) th			F	 			minut	Fundamental mode f = 221.1 C/S
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	4	0	0.25	0.5 6	0.75	-	0	0.25	0.5).75			55	10	75		0	25 0	0.5	.75		0 0	.25 0	5	52.1	-
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	6	0	0.25	0.5	0.75	-	0	0.25	0.5	0.75	-	0	0.25 (0 U.S. 0	.75	-	0 O	.25	0.5 (2.75	-	0	0.25	0.5	0.75	-

the second overarm with its basic model.

Table(.26) Comparison between the fourth modification equipped with

1.33

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1st mode| 1.2

2rdmode 1.02 0 0.87 0.96 0.97 0.97 0.97 0.96 0.95 0.93 0 0.95 0.95 0.96 0.96 0.96 0.96 0.96 0.96 0.94 0 -0.8 2.04 1.62 1.52

aboM	٨										L)efl	ectic	n. r	atic	S										
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1 mode	36.0	0	1.12	1.03	1.01	1.02	1.02	1.03	1.0	1.04	1-04	0	1.03	1.03	1.03	1.03	1.03	1.03	1.03	1.C3	1.03	0	1.16	1.17	1.16	1.16
2 mode	-	0		~		-	-				-	0		-	-	0.96	036	1.01			-	0		11.11	1.05	1.03
3 ^d mode	<u></u>	0		•	-		**			-		0	**	**		*-		9 44		0.99	3 6-0	0	-	0	0.69	0.93
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22270) comparison between the tourth moaitication וביושיים

the third overarm with its basic model.

	m 5	₅ /L ₅	0.5 0.75 1	1.06 1.06 1.06	1.07 1.04 1.02	1 0 83
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٨	ioa - neuc	req:	3	0.97		-
	Mode	number		*t mode	2 ⁿ fmode	3 mode

Table (220) Comparison between the fourth modification equipped with the fourth overarm with its basic model .

-84-

3 rd mode 1 0 1 1 1 0.98 0.98 0.	2 ⁿ mode 1.02 0 0.87 0.96 0.96 0.97 0.97 0.	1 mode 1.2 0 1 0.91 0.89 0.89 0.89 0.	0 0.25 0.5 0.75 1 0 0.		Mode Less Beam 1 E	y
90 1 1	.97 0.96 0.95 0.94	.88 0.86 0.85 0.83	25 0.5 0.7 5 1	X2/L2	3eam 2	
	0 0.95 (0.97 0.95 0.	0 0.86 0.86 0.86 0.	0 0.25 0.5 0.75 1	×1/L3	Beam 3	eflection ratio
11 1.01 1.01 1.01 1.01	960 960 <u>96</u> 0 960 960 96	16 0.36 0.87 0.87 0.87 0.87	0 0.25 0.5 0.75 1	X4/L4	Beam 4	S .
0 0.75 4 1.44 1.3	0 0.8 2 1.6 1.5	0 0.47 0.44 0.42 0.4	0 075 0.5 075 1	X ₅ /L ₅	Beam 5	

iable (. 24) Comparison between the Tourth modification equipped with

the fifth overarm with its basic model.

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Fig. (1)

Main parts and dimensions of the horizontal

knee_type milling machine.







2cale 1:50

50

(1), (2)

The basic model of the machine.









Fig. (4) Mathematical model







(ב) נוא.

Model of the first modification.

-68-









Model of the second modification

-90-









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Model of the third modification.











(OI) . Qif

(6).<u>e</u>if



Fig. (12)

(_{TT}) 614

The idealized models of the four suggested modification





(Trapejoid)





(Rectangle)



The second modification

(Square)

Dimensions in Mm.

Scale 1:1



The fourth modification (Semi-Circular on rectanglar base)

Fig. (13)

Different Cross-Sectional atternative shapes of the overarm.



The third modification

(Semi – Circle)



The fifth modification (Semi-Circular tail)



Fig.(14) Modeforms of the pasic model and the best modifications





(First mode)







(Third mode)







*



(Second mode;



(third mode).

Fig, (22.) The percentage reduction in the deflection of the main parts of the third and modifications





MATHEMATICAL SIMULATION OF MACHINE TOOL STRUCTURES

الترصيف الرياضي لمياكسل آلات الانتساع مستاذ دكتور / عبد المادي ناعر مام ماميد دكتوره / سعاد محد سراع مميندس / سالم مستند هيكل آلة الانتاج هو الجزا الرئيسي لها حيث يساهم في تحديد حالتهسا الديناميكية م لهذا السبب كان من الفروري دراسة أشكال تصبيعية مختلف للهيكل بعساعدة الترصيف الرياضي للميكل م

فى هذا العمل فحست الخسائس الديناميكية لهيكل فريزة أفقية على أسباس طريقة نظرية العتب مثنملا على النبوذج الرياضى للهيكل ، ومساعدة النمسبوذج الرياضى المقدم درس وقورن ١٢ نبوذج مقترح يمثلوا حلولا تصبيعية لها نفسبس الوزن ، الأبعاد ، السمك والكتلسة ،

الدراسة المقدمة بينت امكان توقع الحالة الديناميكية لمعظم الهياكل الملائسة. وبالتالي امكان عمل التحسينات الممكنة لها ٠

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